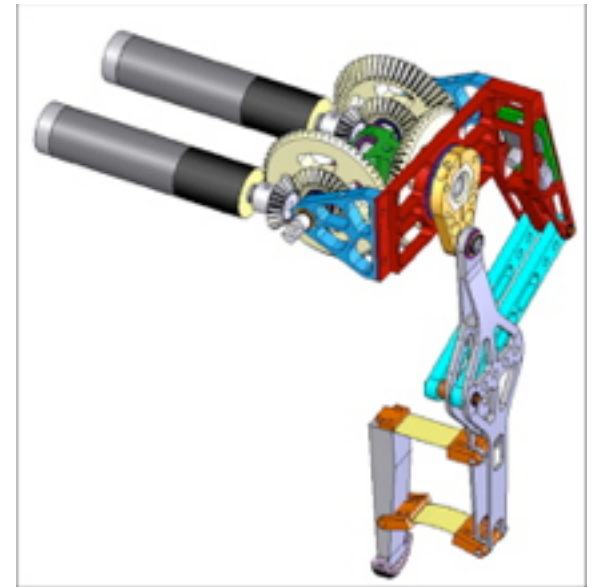


ME112: Mechanical Systems

Lecture 2, Jan 6 2016



Part I:

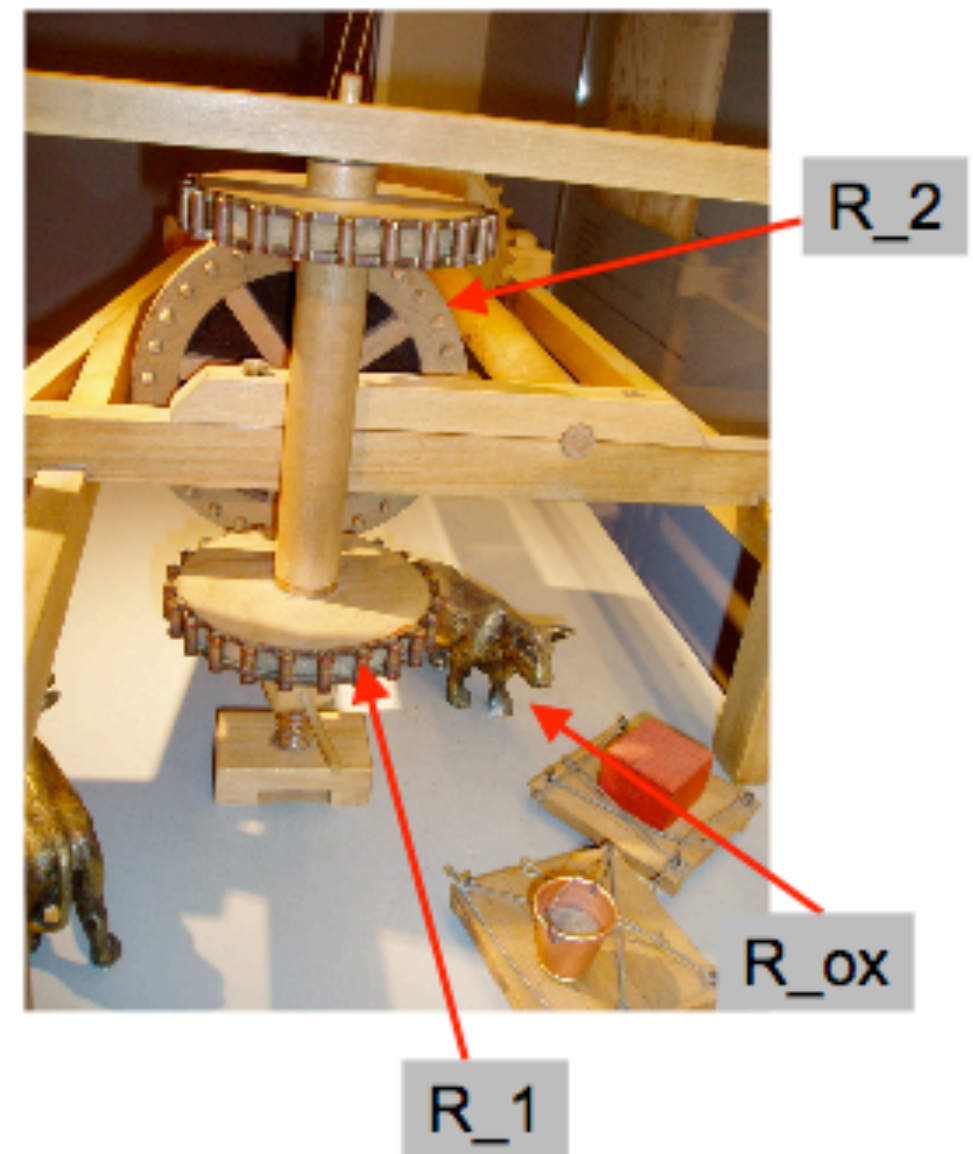
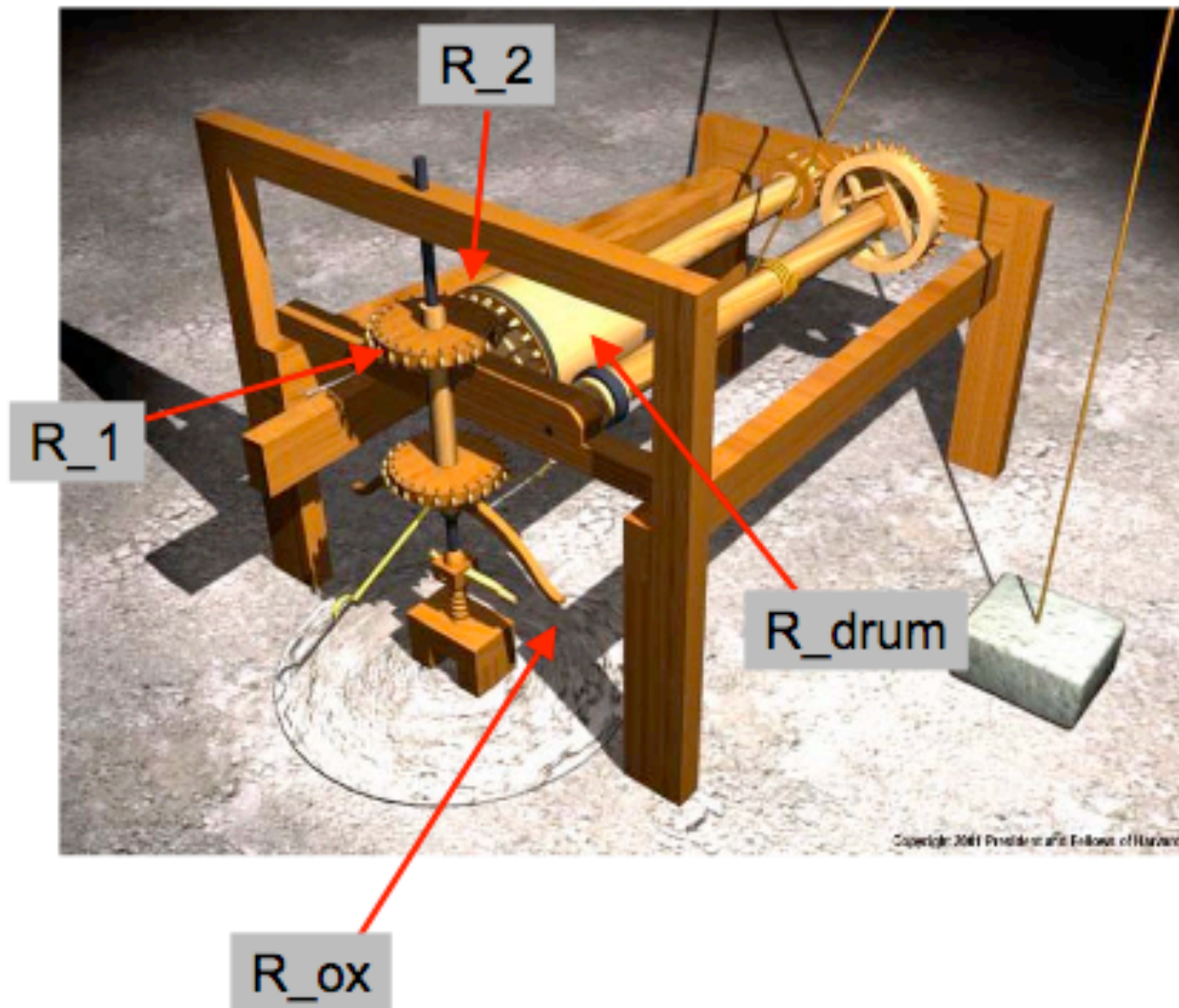
- Lab signups, logistical issues?
- Brunelleschi and his machine
- Assignment 1 questions, discussion

Part II:

- Modern gear kinematics
- Involute profiles
- Contact ratio
- Gear terminology

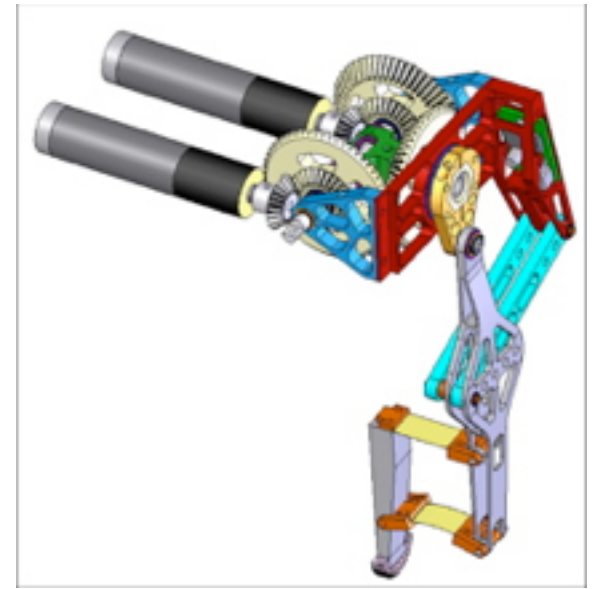
Assignment 1: Read for Wednesday. Think of a couple good questions. We'll talk about it more. Due online Friday.

Example 1: Consider Brunelleschi's giant 3-speed reversing hoist as an example of an input/output system – the same machine used in Assignment 1.



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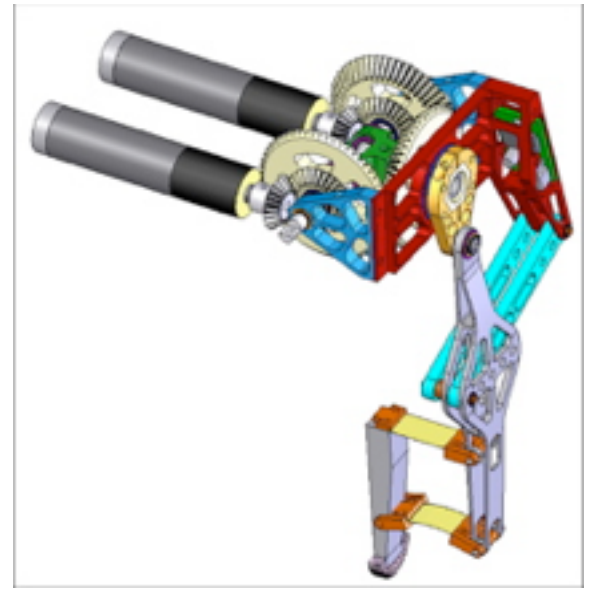
Part I:

- Lab signups, logistical issues?
- Brunelleschi and his machine
- Assignment 1 questions, discussion

Part II:

- Modern gear kinematics
- Involute profiles
- Contact ratio
- Gear terminology

Renaissance vs modern gearing



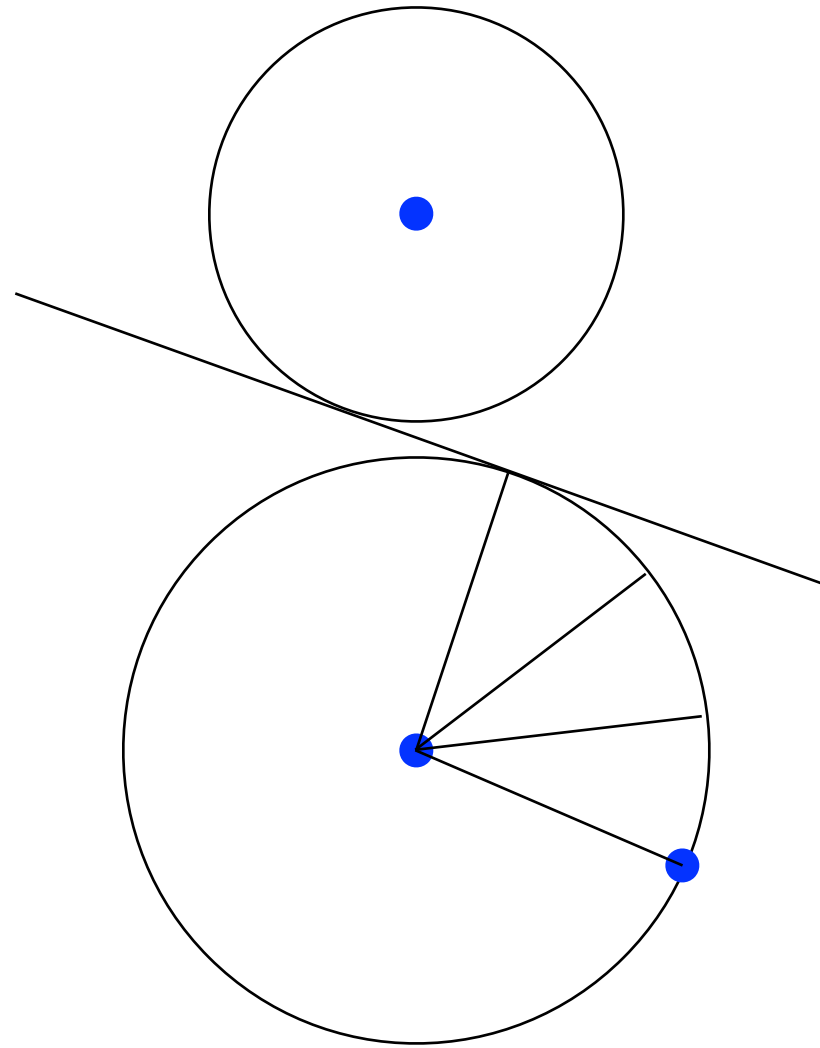
Cog wheel (lantern and peg):

<http://science.howstuffworks.com/transport/engines-equipment/gear1.htm>

Involute:

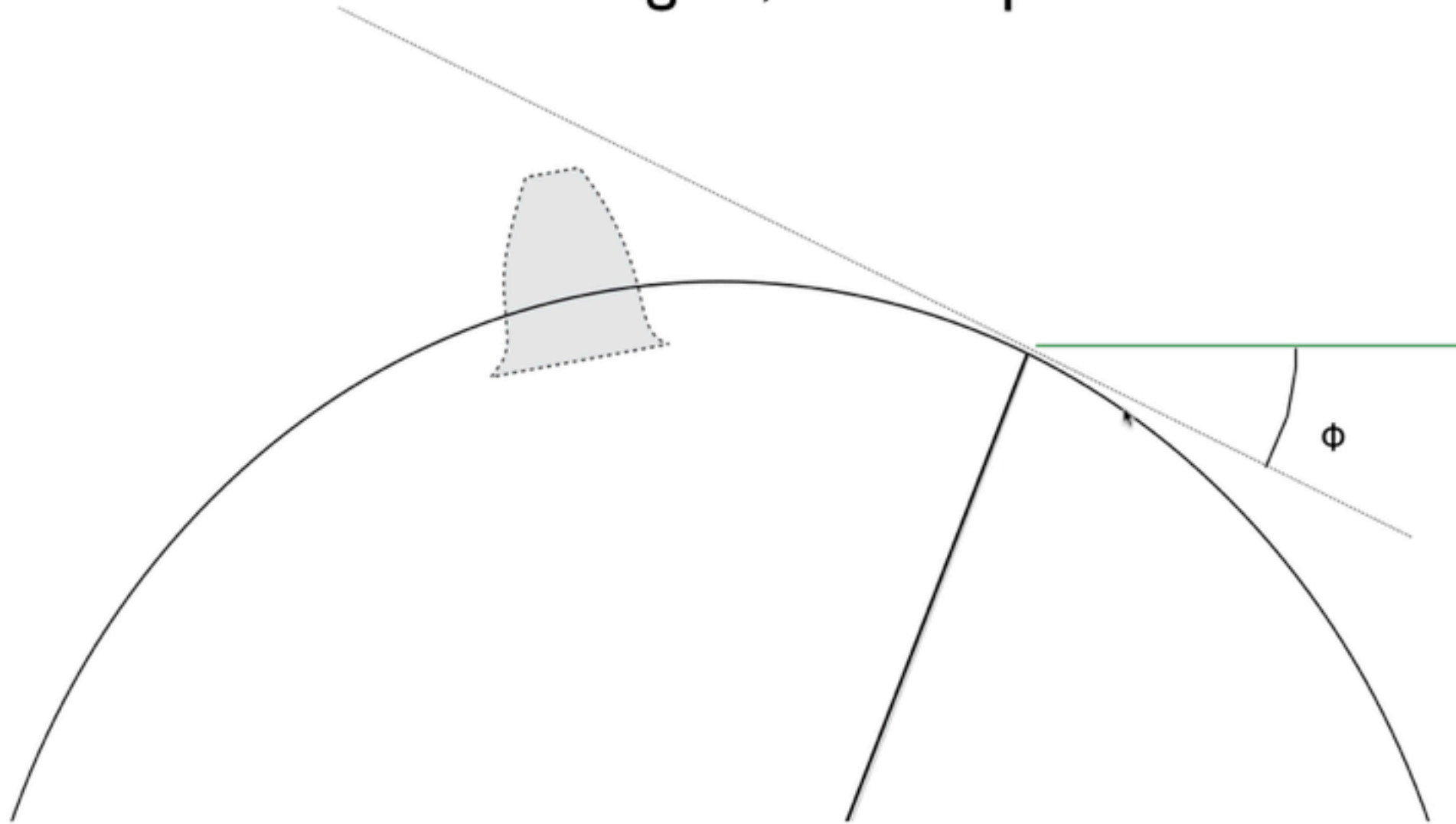
http://www-mdp.eng.cam.ac.uk/web/library/enginfo/textbooks_dvd_only/DAN/gears/meshing/meshing.html

Two rotating cylinders



<https://www.youtube.com/watch?v=4QM0juVXW54>

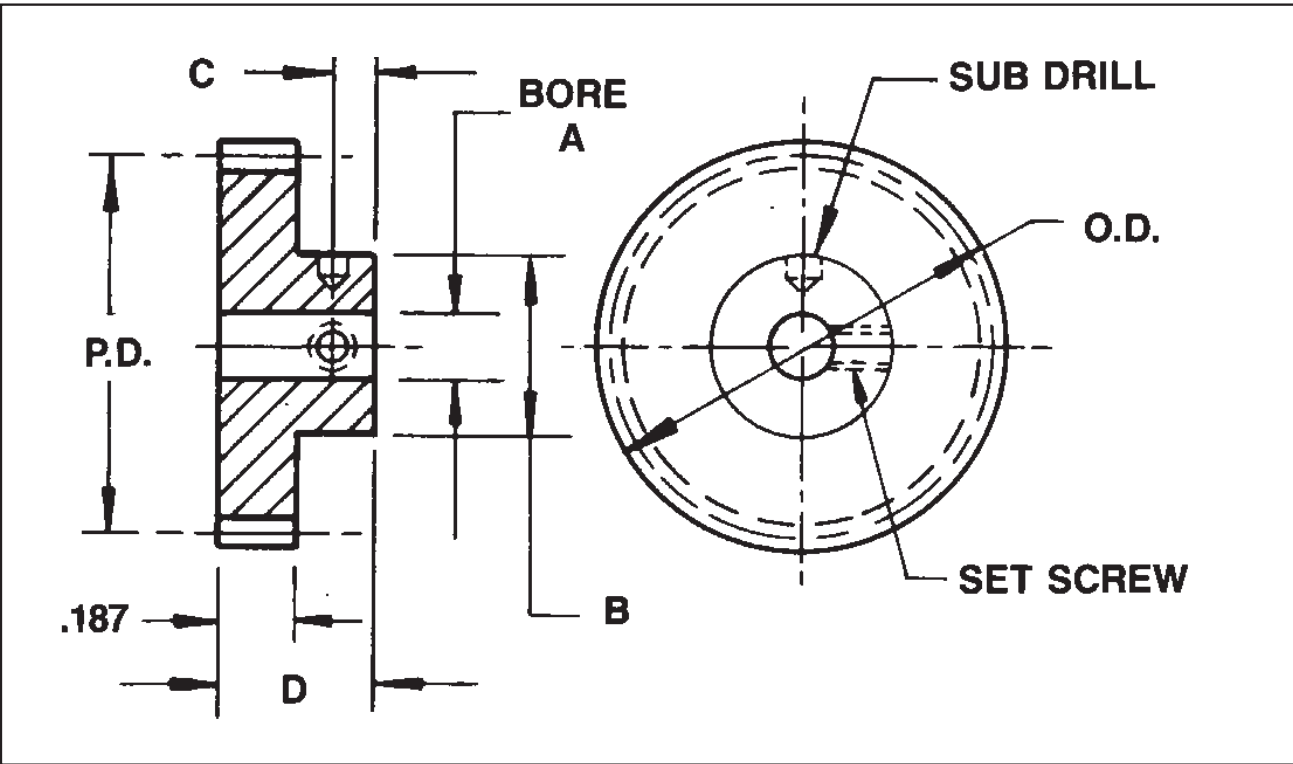
Building the Involute Curve:
consider a reference frame
fixed in the gear, not in space



SPUR GEAR-48 PITCH — 3/16" Face Width ■ 20° Pressure Angle

Pin Hub — 3/16", 1/4" Bores

PIC Gear Catalog



Material: 303 Stainless Steel
2024-T4 Aluminum
(Anodized Before Cutting)

$b = ?$

$P = ?$

$N = ?$

$\phi = ?$

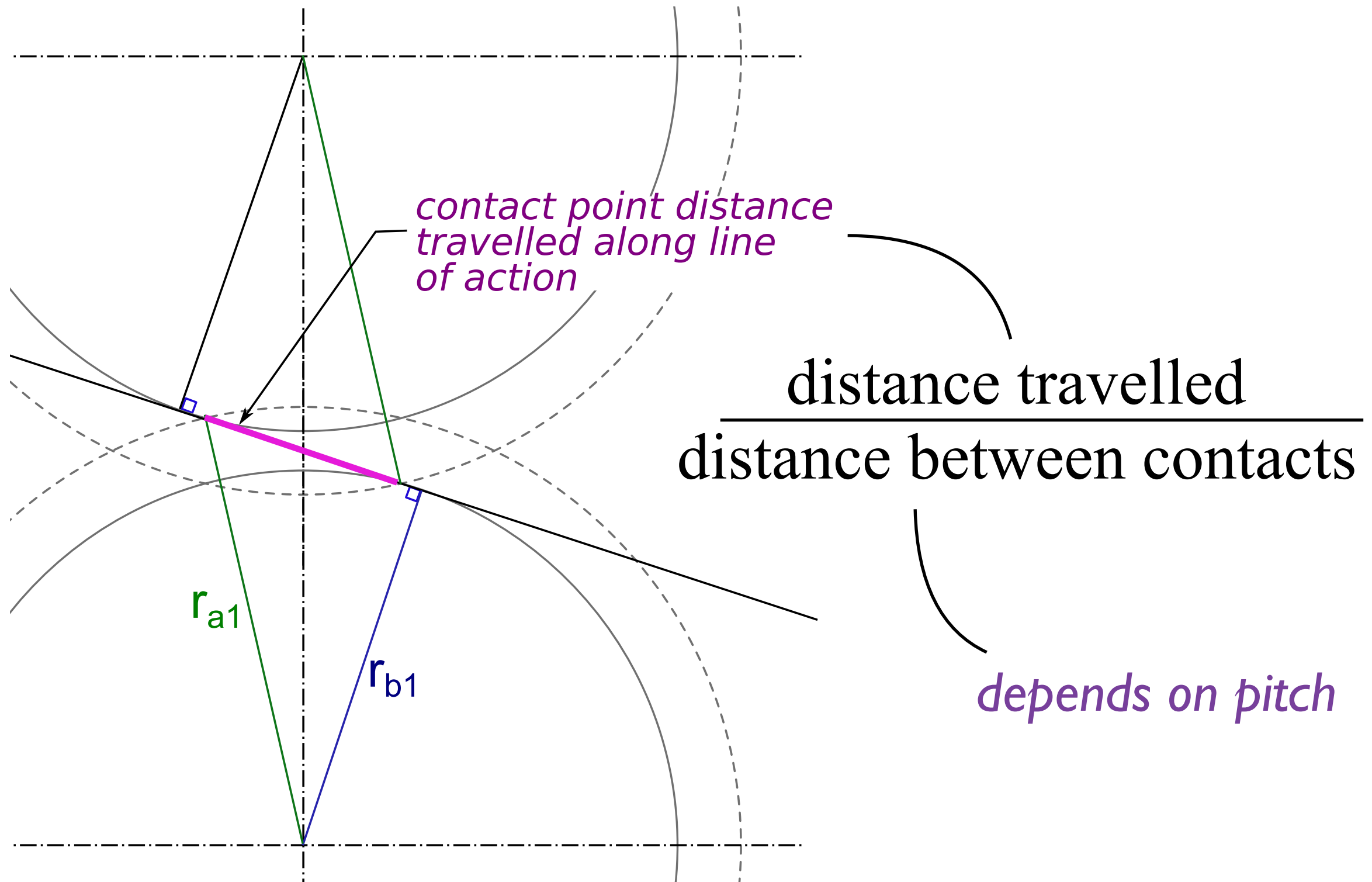
$r_a = ?$ check: $(34+2)/48 = 2 \cdot r_a$

No. teeth	P.D.	O.D.
34	0.7083	0.750

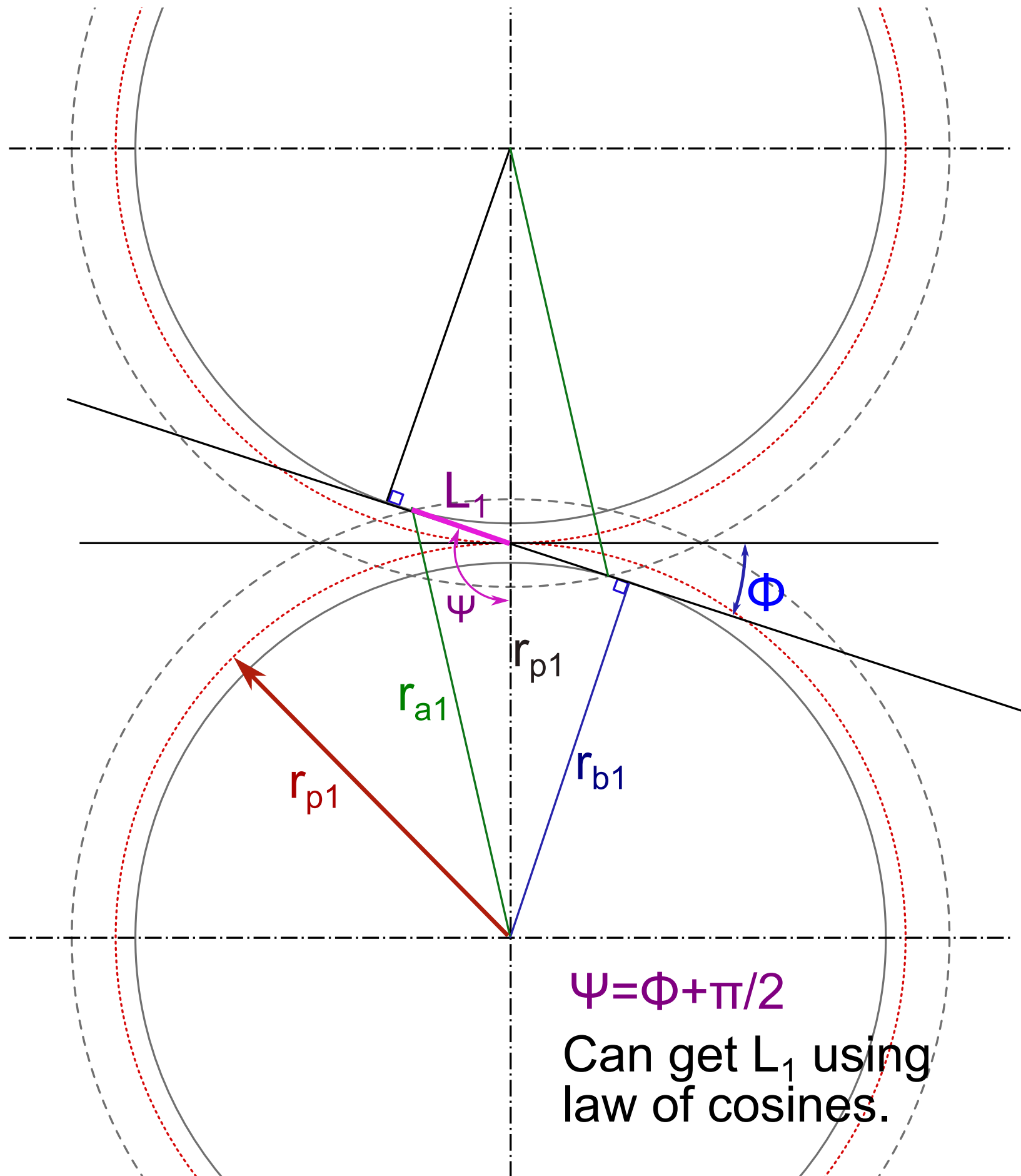
Gear Data			Stainless Steel Part No. Bore Size		Aluminum Part No. Bore Size	
No. Teeth	P.D.	O.D.	.1873	.2498	.1873	.2498
21	.4375	.479	G5-21	—	G6-21	—
22	.4583	.500	G5-22	—	G6-22	—
23	.4791	.521	G5-23	—	G6-23	—
24	.5000	.542	G5-24	—	G6-24	—
25	.5208	.563	G5-25	—	G6-25	—
26	.5416	.583	G5-26	—	G6-26	—
27	.5625	.604	G5-27	G7-27	G6-27	G8-27
28	.5833	.625	G5-28	G7-28	G6-28	G8-28
29	.6041	.646	G5-29	G7-29	G6-29	G8-29
30	.6250	.667	G5-30	G7-30	G6-30	G8-30
32	.6666	.708	G5-32	G7-32	G6-32	G8-32
34	.7083	.750	G5-34	G7-34	G6-34	G8-34
36	.7500	.792	G5-36	G7-36	G6-36	G8-36
38	.7916	.833	G5-38	G7-38	G6-38	G8-38
40	.8333	.875	G5-40	G7-40	G6-40	G8-40
42	.8750	.917	G5-42	G7-42	G6-42	G8-42

Contact ratio

- *How smoothly is load transferred from one pair of teeth to the next?*
- *Do we have more than one pair of teeth in contact at all times?*



Contact ratio



(similar for L_2)

$$\psi = \phi + \pi/2$$

Can get L_1 using
law of cosines.

Lets try it!

Python or Matlab or Spreadsheet...

```
# Some gear pair calculations
import numpy as np #math stuff

Pitch = 24.          # Diametral pitch (teeth per inch of diameter)
phi_deg = 20.        # Pressure angle in degrees
phi = np.pi*(phi_deg/180)

# Gear 1
N1 = 24
dp1 = N1/Pitch;      rp1 = 0.5*dp1      # pitch diameter, radius
da1 = (N1+2)/Pitch;  ra1 = 0.5*da1      # addendum (outer) diameter, radius
db1 = dp1*np.cos(phi); rb1 = 0.5*db1    # base diameter, radius

# Gear 2
N2 = 24
dp2 = N2/Pitch;      rp2 = 0.5*dp2      # pitch diameter, radius
da2 = (N2+2)/Pitch;  ra2 = 0.5*da2      # addendum (outer) diameter, radius
db2 = dp2*np.cos(phi); rb2 = 0.5*db2    # base diameter, radius

speedratio = 1.0*N1/N2 #Force to floating point

# Check contact ratio
C = rp1 + rp2
Pbase = np.pi*np.cos(phi)/Pitch

numerator = np.sqrt(ra1**2-rb1**2) + np.sqrt(ra2**2-rb2**2) - C*np.sin(phi)
denominator = Pbase

ContactRatio = numerator/denominator
# Ideally, ContactRatio > 1.5
```

Lets try it!



Any design rules can be broken,
if you know what you are doing.